# An elasticity-based smoothing post-processing algorithm for the quality improvement of quadrilateral elements 

César Blecua, Antonio Falcó *<br>Departamento de Matemáticas, Física y Ciencias Tecnológicas, Universidad CEU Cardenal Herrera, CEU Universities, San Bartolomé 55, 46115 Alfara del Patriarca, Spain<br>ESI International Chair@CEU-UCH, Universidad Cardenal Herrera-CEU, CEU Universities San Bartolomé 55, 46115 Alfara del Patriarca, Spain

## ARTICLE INFO

## MSC:

00-01
99-00
Keywords:
Mesh quality
Mesh smoothing
Quadrilateral element
Unstructured mesh


#### Abstract

Post-processing meshing algorithms are widely used to achieve the desired quality in quadrilateral meshes. Assuming that the mesh quality depends on the distortion and the size error of each of its convex quadrilaterals, deficiencies arise by considering solutions based in minimizing either the distortion or the size error. To solve this undesirable situation, in this paper we propose a new smoothing post-processing meshing algorithm. This procedure provides a good compromise between the distortion and the size of each element in the mesh. It is formulated by using an elasticity-based argument and allows to be implemented either in sequential or parallel form. Moreover, it provides a good quality output compared with some of the usual smoothing post-processing meshing algorithms.


## 1. Introduction

In CAD a mesh can be seen as the discrete geometric representation of a given physical domain given by a particular set of individual elements such as 2D triangles or quadrilaterals (in 2D geometries) and tetrahedra or hexahedra (in 3D geometries). Thus, considered as a geometric approximation of the physical domain, the mesh quality has a direct impact on the simulation robustness, accuracy and efficiency. Aspects to be considered in order to improve mesh quality are the mesh resolution, mesh geometry, mesh topology, mesh smoothness, the type of elements in the mesh and, if any, the order of the element basis functions. In any case, the practitioner should be concerned with the quality of the mesh to be used. Sometimes, the quality of the mesh is assessed by visual related techniques, that is, by means mesh diagnostics and/or values of quality metrics. Two basic categories of methods for mesh quality improvement can be considered, node movement and mesh topology. Usually, "mesh quality improvement" is often interchanged with "mesh optimization". However, mesh optimization should include a numerical optimization method in which the initial vertex coordinates are changed, and hence a node movement is produced, to minimize a function that also depends on vertex coordinates and is related with a mesh quality measure.

Reinforced by practitioners in engineering and architecture, quadrilateral mesh generation has been a critical research problem in the last decades. However, quadrilateral meshing algorithms provide an output
which - in order to achieve a target quality - usually requires the use of a post-processing operation [1,2]. This procedure that can fall in one of the following two categories. The first one, comprises the socalled clean-up algorithms. They are able to modify the mesh topology by creating, removing, and reconnecting nodes and elements. The second category, constitutes the smoothing algorithms. They proceed by modifying the coordinates of the interior nodes and maintaining the mesh topology. Obviously, by just displacing nodes it is not possible to fix topology issues, thus, if we wish to avoid employing clean-up algorithms, we must take care that the meshing algorithm already provides a proper topology, before the post-processing procedure starts.

Concerning quality measures, two types can be considered, quality measures for isotropic and anisotropic elements. We recall that an isotropic element is defined as a straight-side element for which all angles within a face are equal. Otherwise, we will say that it is a anisotropic element. In particular, in 2D we have the equilateral triangle and the square as isotropic elements. They are used when the application has isotropic physics or is the best element shape for it. Then to quantify the quality of a given element we need to define an ideal element that represents a configuration benchmark (for example, the unit square).

The goal of this paper is to introduce a meshing post-processing algorithm for the quality enhancement of quadrilateral elements. Along this paper we assume that a structured mesh is a mesh whose interior

[^0]nodes, that is, the nodes that are outside the domain's boundary, are the vertices of exactly four sides. Before introducing the post-processing stage, our starting point is the quadrilateral meshing algorithm introduced by Sarrate and Huerta [3]. The aforementioned procedure provides a satisfactory topology (the obtained output is either a structured mesh, when it is used over in a suitable contour, or an unstructured mesh otherwise). This fact allows to propose a smoothing-only postprocessing, avoiding the use of a clean-up method. We also use the following characteristics and assumptions to construct our procedure:

- It does not require an initial triangle mesh (it is a direct method).
- It is a recursive subdivision algorithm. It chooses the best splitting line by assigning a cost line to each possible candidate and then picking the optimal one (hence, it is easy to be implemented).
- It avoids the use of specific treatments with the only exception of 6 sided contours. However, following [4], a simple mapping is performed for subdividing them with some minor modifications.
- We avoid to solve complex geometrical situations (and hence difficult cases that frequently arise in advancing front and paving [5] approaches).

Fig. 1(a) shows an output obtained by a meshing algorithm implemented following the above considerations. It is applied to a two dimensional region with a boundary combining rectangular and circular areas in order to test the behaviour from straight to curved perimeters. Moreover, it has a gradation over the elements size (near the curved border they have a side length around the half of the opposite side straight border). To conclude, observe that the contour is symmetric with respect to the horizontal axis - when the initial contour is symmetric, the procedure provides a symmetric mesh.

The result given in Fig. 1(a) satisfies some of the expected requirements of a quadrilateral mesh with of a topology endowed by square elements: (a) whenever it is possible, it favours a structured meshing, (b) it respects element size gradation when it is variable across the meshing domain, (c) it only uses quadrilateral elements, (d) it works well over arbitrary contours, and (e) the elements tend to inherit the directions of the perimeter (in this example, he quadrilaterals arriving to the circular border acquire a radial directionality). We point out that the above requirements have been achieved without intervention of the end-user.

On the other hand, this mesh is still far from the shape quality that we shall expect. Some quadrilaterals are noticeably distorted, and not all the elements have their size accordingly set to the place they are located within the domain. While the resulting mesh topology is good, some node coordinates need to be modified for achieving the desired quality. More precisely, Fig. 1(b) provides an example of the quality improvement that can result after applying a smoothing postprocessing. Now, it seems to be clear that before proceeding we need to define a performance measure for the "quality" of the mesh topology. Some questions arising from this framework are: Can the "quality" be quantified? if the answer is positive, will it depend on one or in several criteria? and, if it depends on several criteria, will they be mutually diverging?

In a close look at Fig. 1(a), we can observe the following two cases for poor quality elements (Figs. 2(a) and 2(b)):Either they are too distorted (Fig. 2(a)), or have an inadequate size (Fig. 2(b)). Thus, we shall get a mesh with the desired quality by applying a post-processing that fixes these two deficiencies. A natural procedure for improving the quality issues in Figs. 2(a) and 2(b) would be to minimize the distortion of the involved quadrilaterals (we fix the distortion of a square to be zero), which would help reduce the deficiencies in Fig. 2(b); and to minimize the elements' size error, which would address the issues from Fig. 2(a). If we apply a brute force minimization strategy to both approaches (by evaluating a massive number of points), we obtain as output Figs. 3(a)-3(c). In particular, Fig. 3(a) has been obtained by modifying the nodes coordinates under the constraint that the difference between the area of the element and the one for an "ideal

(a) Without post-processing.

(b) With smoothing post-processing.

Fig. 1. Quality improvements from smoothing post-processing. Note that this example has a size gradation for elements (double side length in nodes at the left).


(a) Elements with a size quite different from the
(b) Excessive distortion in some elements. one they should have in
that place.
Fig. 2. Quality deficiencies without post-processing.
square" of the desired size should be minimal. This "ideal square" has a fixed size at each place in the domain. Thus, it allows to address the deficiencies related to the element initial size. However, this approach leads us to an unacceptable distortion of the elements far worse than before the post-processing (compare Fig. 3(a) with Fig. 1(a)).

Now, in Fig. 3(b), we give the result obtained after minimizing a quality measure, called the Oddy distortion (see 2.2.1 below), for each of the quadrilaterals in the mesh. While the mesh looks more pleasant to the eye, a more careful inspection finds that some issues related to wrong sizes of the elements have not been fixed. This is due to the fact that this minimization process ignored the element size. Fig. 3(c) reveals the existence, after the distortion minimization, of quadrilaterals bigger and smaller than their neighbours in places where it seems desirable to have elements of constant size. Therefore, it is not possible to achieve a structured output addressing each of the two aforementioned quality deficiencies separately.

In order to answer the questions given above, we fix that the quality of a quadrilateral mesh depends on optimizing: the distortion and the size of the quadrilaterals. However, the fulfilment of both quantities is in general not possible. Thus, in order to find a quasi-optimal solution we need to set a compromise between both criteria. A first approach to this strategy [3] is a smoothing post-processing algorithm introduced by Giuliani [6]. This procedure tends to minimize the distortion and hence to achieve an uniform element size. However, this last fact has the consequence of blurring, or even losing, the sizes' gradient. Fig. 4(a) shows how the size gradient in the original mesh (Fig. 1(a)) has been noticeably lost when applying the Giuliani algorithm. Indeed, due to this undesirable fact, Sarrate and Huerta proposed in [3] a simple variation for the Giuliani algorithm, trying to respect the size of the mesh elements. Obviously this cannot fix the size errors that already were in the mesh before post-processing (in fact this variation of the algorithm has no knowledge of the desired element size, so it cannot make any size corrections), but it does better respect size gradations (Fig. 4(b)).

We must mention however that experience shows this variation of the Giuliani algorithm not only respects the size gradations, but tends to exaggerate them, introducing visually noticeable noise patterns throughout the mesh because it produces areas with elements smaller than the average size in their neighbourhood. At Fig. 4(b), it can be seen that this algorithm not only has not made the elements more uniformly sized, but it has even incremented the differences, and these noise patterns can be perceived. In [7], the authors introduce a new proposal for smoothing post-processing based on the distortion formulation by Knupp [8]. In this new development, the element size error is included in the function to be minimized, so that the post-processing not only reduces distortion, but also the size error. Minimization on each element, denoted by elem, is performed by a function which is the product of the distortion of the given element, denoted by $\eta_{s h}$ and where $1 \leq \eta_{s h}($ elem $)<\infty$, by the size error, denoted by $\eta_{s i}$ and where $1 \leq \eta_{\text {si }}($ elem $\left.)<\infty\right)$, i.e.
$\eta($ elem $)=\eta_{\text {sh }}($ elem $) \eta_{s i}($ elem $)$.
Now, we can understand the idea we outlined before: seeking optimal quality implies a compromise between distortion and size error (in this case, the compromise has been chosen as the product between the two). The development in [7] is based in the distortion formulation by Knupp [8] and it minimizes the arithmetic mean (rather than the maximum) of the distortion at the element vertices, and because of this it cannot be directly compared to the results of the algorithm that we propose. Thus, we have made a reinterpretation of [7], implementing it with the same distortion formulation that our algorithm uses, in order to compare it with our results. The output from this reinterpretation is shown in Fig. 4(c). Considering that the elements are quadrilaterals, it is obvious that their diagonals have a great responsibility on the quadrilateral distortion, while the sides affect mainly its size. Of course these are not completely independent responsibilities: diagonals

(a) Minimization of the element size error.

(b) Minimization of the Oddy distor-
tion.

(c) Close-up of the Oddy distortion minimization, showing elements notably bigger and smaller than their neighbours (in a place where all elements should have the same size).

Fig. 3. First approaches towards quality improvement.
have also an impact on the quadrilateral size (keeping the distortion constant, diagonals will be longer in a bigger quadrilateral), and the side lengths also affect distortion (the more uniform the side lengths, the more likely that it will be possible to achieve a low distortion). But it is clear that if we keep the sides with a fixed length, the diagonals

(a) Original Giuliani algorithm.

(b) Giulani algorithm as modified by

Sarrate/Huerta.

(c) Minimization of the product of the size error by the average of the distortion.

Fig. 4. Some smoothing post-processing algorithms.
get a crucial importance in distortion (see Fig. 5). And if we wish to modify the size without altering the distortion, we need to apply the same scale factor to all the sides and the diagonals.

To conclude, other two approaches can be used to improve the mesh quality and the element size. Aparicio-Estrems, Gargallo-Peiró and Roca [9] introduce a quality measure for isotropic 2 D linear elements and constant metric that can be used to optimize the location of the nodes to minimize the element distortion [10]. The objective function used in [10] depends of the whole set of individual elements. Thus, its parallel implementation appears as a complicated task.

The target matrix optimization paradigm (TMOP) [11] allows to improve the mesh quality and the element size by using a local quality metrics together a set of previously created target matrices. However, there is not yet a general method to create correct and effective goals in response to a specific mesh quality improvement target.


Fig. 5. The sides of both quadrilaterals have the same length, but their diagonals vary, altering the distortion.


Fig. 6. Equilibrium at a point.

The paper is organized as follows. In Section 2 we introduce the proposal elasticity-based smoothing algorithm for a quadrilateral mesh depending on a individual smoothing procedure function. Section 3 is devoted to the numerical implementation and also some computational results will be provided. Finally, in Section 4, some conclusions and final remarks are given.

## 2. A elasticity-based smoothing algorithm for a quadrilateral mesh

The aim of this section is to give some preliminary definitions that we will use to describe our quadrilateral mesh smoothing algorithm for a given two dimensional region and also to explain the smoothing strategy used in each vertex of the mesh. As we said above, we will be taking a compromise between the lengths of the sides and the diagonals over each quadrilateral in the mesh. Formulating a balance of lengths suggests the physical parallel of a spring system, or the equilibrium of members subjected to axial force. There is previous work in springbased smoothing post-processing (Lohner et al. [12], as well as an implementation in the ANSYS software), but those developments do not take the quest for achieving minimum distortion in quadrilateral elements (their formulation assumes triangle meshes). This will require to introduce a distortion magnitude into the model, so that the diagonals tend to minimize it. Moreover, it shall be also necessary to take into account the quadrilateral sides in order to reach the desired compromise between size and distortion.

Let us consider a quadrilateral mesh $\mathfrak{Q}(\Omega)=\left\{\mathcal{Q}_{\alpha}: \alpha \in \mathfrak{I}\right\}$ of a given two dimensional connected region $\Omega$. We will consider the set of mesh points given by the vertices of the individuals in $\mathfrak{Q}(\Omega)$, that is,
$\mathcal{V}(\mathfrak{Q}(\Omega))=\{\boldsymbol{P} \in \Omega: \boldsymbol{P}$ is a vertex of $\mathcal{Q}$ for some $\mathcal{Q} \in \mathfrak{Q}(\Omega)\}$.

Next, we will explain for a generic fixed point $P \in \mathcal{V}(\mathfrak{Q}(\Omega))$ (just before the smoothing post-processing) the strategy for computing its smoothed version. Let $\boldsymbol{P}_{\boldsymbol{S}}$ be the displaced position for $\boldsymbol{P}$ after the smoothing post-processing. To compute $\boldsymbol{P}_{S}$, we assume that $\left\{\boldsymbol{P}_{i} \in \mathcal{V}(\mathfrak{Q}(\Omega)): 1 \leq i \leq n\right\}$ are the initial $n$ mesh points connected to $\boldsymbol{P}$ by either a side or a diagonal of a quadrilateral $\mathcal{Q} \in \mathfrak{Q}(\Omega)$. More precisely, given $\boldsymbol{P}$ we will take into account the set
$\mathcal{C}(\boldsymbol{P}):=\left\{\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}: \boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}\right.$ is either a side or a diagonal
for some $\mathcal{Q} \in \mathfrak{Q}(\Omega), 1 \leq i \leq n\}$.
(see Fig. 6). Let denote by $n_{s}$ (respectively, $n_{d}$ ) the number of sides in $\mathcal{C}(\boldsymbol{P})$ (respectively, the number of diagonals in $\mathcal{C}(\boldsymbol{P})$ ). Thus, $n=n_{s}+n_{d}$. We also assume the existence of a map
$L: \mathcal{C}(\boldsymbol{P}) \longrightarrow(0, \infty), \quad \boldsymbol{P}_{i} \boldsymbol{P} \mapsto L\left(\boldsymbol{P}_{i} \boldsymbol{P}\right):=L_{i}$,
where $L_{i}$ is a previously fixed ideal length for $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}$ (in sections 2.1 and 2.2 below we explain the practical construction of $L$ ).

Let $\boldsymbol{v}_{i}:=\left(v_{i}^{x}, v_{i}^{y}\right)$ be direction vector associated to the corresponding segment $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}$. Thus, $\left\|\boldsymbol{v}_{\boldsymbol{i}}\right\|$ is the length of $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}$. Let $\boldsymbol{t}=\left(t^{x}, t^{y}\right)$ be the displacement vector $\boldsymbol{P}_{\boldsymbol{S}} \boldsymbol{P}$ joining $\boldsymbol{P}$ together its smoothed position, $\boldsymbol{P}_{\boldsymbol{S}}$. It will be our goal variable that needs to be calculated (see also Fig. 6).

In order to define an optimal value for $\boldsymbol{t}$, we consider that $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}$ is a bar with a length $L_{i}$ and associated cross sectional area $A$. We assume that $L_{i}$ is given and $A$ is a fixed constant independent of $i$ that needs to be determined. Consider that $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}$ is subjected to equal and opposite axial forces $N_{S, i}$ pulling at the ends when the bar is under tension. Then we assume that $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}$ suffers a stress producing a distortion length $\left\|\boldsymbol{v}_{\boldsymbol{i}}+\boldsymbol{t}\right\|$. In consequence, the strain will be given by the ratio:
$\frac{\left\|v_{i}+t\right\|-L_{i}}{L_{i}}$.
Let $E_{i}(t)$ be a given non-linear function of $t$ that represents a modulus of elasticity for $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P} \in \mathcal{C}(\boldsymbol{P})$ (it will be different depending if $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P}$ is either a side or a diagonal and we will give its construction below). Now, following the relationship between stress-strain and the modulus of elasticity, we can write
$\frac{N_{S, i}}{A}=E_{i}(t) \frac{\left\|v_{i}+t\right\|-L_{i}}{L_{i}}$,
We consider that the direction vector representing the axial force direction after smoothing is:
$u_{S, i}=\frac{v_{i}+t}{\left\|v_{i}+t\right\|}$
and, hence the $i$ th axial force after smoothing exerted at point $\boldsymbol{P}_{S}$, yields:
$F_{i}=u_{S, i} \cdot N_{S, i}=\frac{v_{i}+t}{\left\|v_{i}+t\right\|} \cdot \frac{\left(\left\|v_{i}+t\right\|-L_{i}\right) E_{i}(t) A}{L_{i}}$.
Assuming an equilibrium of forces at the smoothed point $\boldsymbol{P}_{S}$ :
$\sum_{i=1}^{n} \boldsymbol{F}_{i}=\mathbf{0}$,
leads to:

$$
\begin{equation*}
\sum_{i=1}^{n} \frac{v_{i}+t}{\left\|v_{i}+t\right\|} \cdot \frac{\left(\left\|v_{i}+t\right\|-L_{i}\right) E_{i}(t)}{L_{i}}=\mathbf{0} \tag{6}
\end{equation*}
$$

which is a system of equations in $t$ (independent of the constant $A$ ). In order to solve (6) we consider the following Individual Smoothing Procedure (ISP) function
$\boldsymbol{f}_{\boldsymbol{P}}(\boldsymbol{t}):=\sum_{i=1}^{n} \boldsymbol{f}_{\boldsymbol{P}}^{(i)}(\boldsymbol{t})=\sum_{i=1}^{n}\left(f_{1}^{(i)}(\boldsymbol{t}), f_{2}^{(i)}(\boldsymbol{t})\right)$.
where
$f_{P}^{(i)}(t):=\frac{v_{i}+t}{\left\|v_{i}+t\right\|} \cdot \frac{\left(\left\|v_{i}+t\right\|-L_{i}\right) E_{i}(t)}{L_{i}}$ for $1 \leq i \leq n$.

Observe, that we can write the above function as
$f_{P}^{(i)}(t)=g_{P}^{(i)}(t) E_{i}(t)$
where
$g_{P}^{(i)}(t):=\left(v_{i}+t\right) \cdot \frac{\left(\left\|v_{i}+t\right\|-L_{i}\right)}{L_{i}\left\|v_{i}+t\right\|}$,
and we have the following lemma.
Lemma 2.1. Given $\boldsymbol{v}_{\boldsymbol{i}} \in \mathbb{R}^{2}$ consider the open set $\mathcal{O}_{i}=\left\{\boldsymbol{t} \in \mathbb{R}^{2}: \boldsymbol{t}+\boldsymbol{v}_{\boldsymbol{i}} \neq\right.$ $\mathbf{0}\}$. Then the function $g_{P}^{(i)}: \mathcal{O}_{i} \longrightarrow \mathbb{R}^{2}$ is $\mathcal{C}^{1}\left(\mathcal{O}_{i}\right)$ and its derivative is given by

$$
\begin{aligned}
D \boldsymbol{g}_{\boldsymbol{P}}^{(i)}(\boldsymbol{t})= & \frac{\left(\left\|\boldsymbol{v}_{\boldsymbol{i}}+\boldsymbol{t}\right\|-L_{i}\right)}{L_{i}\left\|\boldsymbol{v}_{\boldsymbol{i}}+\boldsymbol{t}\right\|}\left(\begin{array}{cc}
1 & 0 \\
0 & 1
\end{array}\right) \\
& +\frac{1}{\left\|\boldsymbol{v}_{\boldsymbol{i}}+\boldsymbol{t}\right\|^{3}}\left(\begin{array}{cc}
\left(v_{i}^{x}+t^{x}\right)^{2} & \left(v_{i}^{x}+t^{x}\right)\left(v_{i}^{y}+t^{y}\right) \\
\left(v_{i}^{x}+t^{x}\right)\left(v_{i}^{y}+t^{y}\right) & \left(v_{i}^{y}+t^{y}\right)^{2}
\end{array}\right)
\end{aligned}
$$

Proof. See Appendix A.
Next, we decompose the map $f_{P}$ as follows. Let
$\boldsymbol{f}_{\boldsymbol{P}}(\boldsymbol{t})=\boldsymbol{f}_{\boldsymbol{P}}^{(s)}(\boldsymbol{t})+\boldsymbol{f}_{\boldsymbol{P}}^{(d)}(\boldsymbol{t})=\sum_{j=1}^{n_{s}}\left(f_{1}^{(j)}(\boldsymbol{t}), f_{2}^{(j)}(\boldsymbol{j})\right)+\sum_{k=1}^{n_{d}}\left(f_{1}^{(k)}(\boldsymbol{t}), f_{2}^{(k)}(\boldsymbol{t})\right)$,
where the $\operatorname{map} \boldsymbol{f}_{\boldsymbol{P}}^{(s)}(\boldsymbol{t})$ takes into account the sides in $\mathcal{C}(\boldsymbol{P})$ and the map $\boldsymbol{f}_{\boldsymbol{P}}^{(d)}(\boldsymbol{t})$ takes into account the diagonals. Observe that from (8)-(9) we write
$f_{P}^{(i)}(t)=\frac{\left(v_{i}+t\right)}{L_{i}} \cdot\left(1-\frac{L_{i}}{\left\|v_{i}+t\right\|}\right) \cdot E_{i}(t)$.
Thus, we will consider $E_{i}(t)=1+\widetilde{\Theta}_{i}(t)$ where $\widetilde{\Theta}_{i}(t)$ is a non-negative function that represents the deviation with respect an "ideal" quadrilateral. In particular, we will take $\widetilde{\Theta}_{i}(t)$ for the quadrilateral sides as a function depending on the quantity $1-\frac{L_{i}}{\left\|v_{i}+t\right\|}$ (which takes zero value when $\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|=L_{i}$ ) and for the quadrilateral diagonals as a function depending on the Oddy distortion value.

Next we give the $L_{i}$ values and the functions $E_{i}$ used to construct $\boldsymbol{f}_{\boldsymbol{P}}$. In particular, Section 2.1 is devoted to the case for the quadrilateral sides and Section 2.2 for the quadrilateral diagonals.

### 2.1. The individual smoothing procedure function over the quadrilateral sides

As we discussed earlier, the quadrilateral sides have their part of responsibility for achieving the desired element size. In this situation we assume that for each $P \in \mathcal{V}(\mathfrak{Q}(\Omega))$ there exists a previously fixed ideal length, denoted by $\ell_{\boldsymbol{P}}>0$. Then, for each side $\boldsymbol{P}_{\boldsymbol{i}} \boldsymbol{P} \in \mathcal{C}(\boldsymbol{P})$ we will define
$L_{i}=\frac{\ell_{\boldsymbol{P}_{\boldsymbol{i}}}+\ell_{\boldsymbol{P}}}{2}$.
Now, we model the modulus of elasticity for the quadrilateral sides as
$E_{i}(\boldsymbol{t})=1+e^{c\left(\left|1-\frac{L_{i}}{\left\|\boldsymbol{v}_{i}+t\right\|}\right|-d\right)}$
where $c, d$ are parameters for adjusting the exponential function as appropriate (our software implementation uses $c=1$ and $d=0$ with acceptable results).

Therefore, the elements of the Jacobian matrix $D f_{P}^{(s)}(t)$ corresponding to quadrilateral sides can be computing by using that
$D f_{P}^{(i)}(t)=D g_{P}^{(i)}(t) E_{i}(t)+g_{P}^{(i)}(t) D E_{i}(t)$,
Lemma 2.1 and together some algebra to obtain:
$\frac{\partial f_{1}^{(i)}(\boldsymbol{t})}{\partial t^{x}}=\frac{\left[\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|^{3}+r-L_{i}\left(v_{i}^{y}+t^{y}\right)^{2}\right] \cdot E_{i}(\boldsymbol{t})-r}{L_{i}\left\|\boldsymbol{v}_{i}+t\right\|^{3}}$,
$\frac{\partial f_{1}^{(i)}(\boldsymbol{t})}{\partial t^{y}}=\frac{\partial f_{2}^{(i)}(\boldsymbol{t})}{\partial t^{x}}=\frac{\left[E_{i}(\boldsymbol{t})+s\left(E_{i}(\boldsymbol{t})-1\right)\right]\left(v_{i}^{x}+t^{x}\right)\left(v_{i}^{y}+t^{y}\right)}{\left\|\boldsymbol{v}_{\boldsymbol{i}}+\boldsymbol{t}\right\|^{3}}$,
$\frac{\partial f_{2}^{(i)}(\boldsymbol{t})}{\partial t^{y}}=\frac{\left[\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|^{3}+q-L_{i}\left(v_{i}^{x}+t^{x}\right)^{2}\right] \cdot E_{i}(\boldsymbol{t})-q}{L_{i}\left\|\boldsymbol{v}_{i}+t\right\|^{3}}$,
where
$s=c\left|1-\frac{L_{i}}{\left\|\boldsymbol{v}_{i}+t\right\|}\right|, \quad ; \quad q=L_{i} s\left(v_{i}^{y}+t^{y}\right)^{2}, \quad ; \quad r=L_{i} s\left(v_{i}^{x}+t^{x}\right)^{2}$,
and $1 \leq i \leq n_{s}$. The non-linearity of $E_{i}(t)$ will be essential as we want members whose length is farther from their goal length, to react with exponentially growing axial force (if $E_{i}$ was linear, a very deformed member would not be able to counteract a big set of better sized neighbouring members, leading to situations where badly shaped elements appear surrounded by a crowd of well shaped elements: non-linear behaviour avoids those situations).

### 2.2. The individual smoothing procedure function over the quadrilateral diagonals

In the previous section, we supposed that the goal length for the diagonals, $L_{i}$ is previously known. The different impact on the quality of a quadrilateral of its sides versus its diagonals, allows us to fix a compromise between the quadrilateral distortion and size. This fact invites to formulate a compromise between the lengths of the sides and the diagonals by means the notion of the distortion of a quadrilateral element. There are several proposals for its quantification and, in particular, Oddy et al. [13] suggest a simple approach that can be applied to parallelograms and which can be extended to arbitrary quadrilaterals. Lee and Lo [14] use a distortion measure to get well shaped quadrilaterals by merging triangles (their algorithm generates quadrilaterals by grouping triangles from an already existing mesh). Canann et al. [15] use the Lee and Lo approach, with some modifications, for their implementation in the ANSYS software. Knupp [8] proposes a general formulation by considering a set of properties that affect the elements' quality. Amongst all the available distortion evaluation formulations, we have chosen the one by Oddy et al.. This choice is motivated for the sake of having an easy implementation and for its good results at assessing quadrilaterals quality. However, the algorithm we described in the previous section is independent on it, and can be applied by using other distortion evaluation approaches. In this paper, the use of the Oddy distortion measure allows to give a collection of simplified equations that can be easily implemented in a computer programme. Next, we will introduce the definition of the Oddy distortion measure for a quadrilateral.

### 2.2.1. The Oddy distortion measure for a quadrilateral

Following [16], the Oddy distortion of a given parallelogram $\mathcal{P}$ is defined by
$D_{\text {Oddy }}(\mathcal{P})=2\left(Q_{\text {Oddy }}^{2}(\mathcal{P})-1\right)$,
here $Q_{\text {Oddy }}(\mathcal{P})$ is the geometrical efficiency:
$Q_{\text {Oddy }}(\mathcal{P})=\frac{l_{1}^{2}+l_{2}^{2}}{2 A}$,
and where $l_{1}$ and $l_{2}$ are the lengths of any of the two adjacent sides in $\mathcal{P}$, and $A$ is its area.

A way for generalizing this expression to arbitrary quadrilaterals consists in evaluating it at the four parallelograms that can be constructed from each of the vertices in the quadrilateral $\mathcal{Q}$ (drawing parallel lines to the both sides meeting at the vertex). In [16], this leads an expression for the Oddy distortion at a given vertex in an arbitrary quadrilateral. Let $\left(x_{j}, y_{j}\right)$ be the coordinates of the $j$ th vertex of a given local quadrilateral $\mathcal{Q}$, and put
$[i]=k$ for some $1 \leq k \leq 4$ if and only if $[i] \equiv k \bmod 4$.

Then the distortion for each vertex $i=1,2,3,4$ is computed by considering
$l_{[i]}^{2}=\left(x_{[i+1]}-x_{[i]}\right)^{2}+\left(y_{[i+1]}-y_{[i]}\right)^{2}$
$l_{[i+3]}^{2}=\left(x_{[i+3]}-x_{[i]}\right)^{2}+\left(y_{[i+3]}-y_{[i]}\right)^{2}$
$A_{[i]}=\left(x_{[i+1]}-x_{[i]}\right)\left(y_{[i+3]}-y_{[i]}\right)-\left(x_{[i+3]}-x_{[i]}\right)\left(y_{[i+1]}-y_{[i]}\right)$
$Q_{\mathrm{Oddy}, i}(\mathcal{Q})=Q_{\mathrm{Oddy}, i}\left(x_{[i+3]}, x_{[i+1]}, x_{[i]}, y_{[i+3]}, y_{[i+1]}, y_{[i]}\right)=\frac{l_{[i]}^{2}+l_{[i+3]}^{2}}{2 A_{[i]}}$
$D_{\mathrm{Oddy}, i}(\mathcal{Q})=2\left(Q_{\mathrm{Oddy}, i}^{2}(\mathcal{Q})-1\right)$.
Observe that the function $Q_{\text {Oddy }, i}$ is $\mathcal{C}^{2}$-differentiable in the open set $U_{[i]}:=\left\{\left(x_{[i+3]}, x_{[i+1]}, x_{[i]}, y_{[i+3]}, y_{[i+1]}, y_{[i]}\right) \in \mathbb{R}^{6}: A_{[i]} \neq 0\right\}$,
and hence also $D_{\text {Oddy }, i}(\mathcal{Q}) \in C^{2}\left(U_{[i]}\right)$. Given that we want a single scalar value for measuring the global distortion of the quadrilateral, the means for avoiding neglecting local shape problems is to take the maximum distortion value amongst all the vertices in the quadrilateral. Thus,
$D_{\text {Oddy }}(\mathcal{Q})=\max _{1 \leq i \leq 4}\left(D_{\text {Oddy }, i}(\mathcal{Q})\right)$,
is non-differentiable, and therefore computing a minimum value by classical smooth methods is not possible. The solution in [16] for having a differentiable function is to use the average value from the distortions at the four vertices. However, as the authors point out, this can result in a distortion value lower than the one that should be assigned to the quadrilateral. In our approach, this is not necessary as we explain below, which allows us to consider a function based on the maximum rather than on the average. The map $D_{\text {Oddy }}(\mathcal{Q})$ is in fact a piecewise $\mathcal{C}^{2}$-function in $\mathbb{R}^{8}$ (see Chapter 4 in [17]) by considering that each $D_{\text {Oddy }, i}(\mathcal{Q})$ is defined over $\left(x_{1}, x_{2}, x_{3}, x_{4}, y_{1}, y_{2}, y_{3}, y_{4}\right) \in \mathbb{R}^{8}$ for $1 \leq i \leq 4$. In consequence, the $D_{\text {Oddy }}(\mathcal{Q})$ is a semi-smooth function (see Section 2.5.3 in [18]) and hence a non-smooth version of the classical Newton method is available (see Chapter 3 in [18]).

This strategy helps us to detect local anomalies that appear at some vertices. On the other hand, a direct implementation of Eq. (12) can generate asymptotes (in the case when $A=0$ ) or provide relatively low distortion values in degenerate quadrilaterals (in the case that the quadrilateral has concave vertices). This last possibility is due to the fact that raising $Q_{\text {Oddy }}$ to two makes it lose its sign, which is equal to the sign of $A$ (note that a vertex is concave if and only if the area $A$ of the local parallelogram at the vertex is negative (assuming counterclockwise vertex order, a requirement that we impose to construct our meshes). These cases are used in [16] to propose modifications to the $D_{\text {Oddy }}$ definition. However, our algorithm imposes that $A>0$ for all local parallelograms, which allows us to consider the original formulation for $D_{\text {Oddy }}$.

Next we will provide an analytical formulation for computing the diagonal goal length $L_{i}$ with respect to a fixed vertex. This value, as we will explain below, will be optimal for the Oddy distortion measure.
2.2.2. An optimal diagonal length $L_{i}$ associated to a quadrilateral $\mathcal{Q}$ with respect to a fixed vertex

Let $\mathcal{Q}$ be a quadrilateral with vertices $\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}$ in counter-clockwise order, namely $\mathcal{Q}=\left\{\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}$ (see Fig. 7). Assume that $\boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}$ are fixed points and we wish to construct a point $\boldsymbol{P}_{\boldsymbol{O}}$ displacing $\boldsymbol{P}$ in the diagonal direction $\boldsymbol{P} \boldsymbol{P}_{\boldsymbol{i}}$ such that the quadrilateral $\delta \mathcal{Q}=\left\{\boldsymbol{P}_{\boldsymbol{O}}, \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}$ satisfies $D_{\text {Oddy }}(\delta \mathcal{Q})<D_{\text {Oddy }}(\mathcal{Q})$. Our goal is to obtain a quadrilateral $\delta \mathcal{Q}$ with a diagonal length $L_{i}=\left\|\boldsymbol{P}_{\boldsymbol{O}} \boldsymbol{P}_{\boldsymbol{i}}\right\|$ for which the $D_{\text {Oddy }}(\delta \mathcal{Q})$ is a minimum. More precisely, we prove the following proposition.


Fig. 7. Quadrilateral with $\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}$ vertices in counterclockwise order. Point $\boldsymbol{P}$ is the new position for $\boldsymbol{P}_{O}$ on the diagonal $\boldsymbol{P} \boldsymbol{P}_{i}$.


Fig. 8. Smoothing algorithm. The springs considered at the equilibrium in A have been drawn with a wider width.

Proposition 2.2. Let $\mathcal{Q}$ be a quadrilateral with vertices $\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}$ in counter-clockwise order, denoted as $\mathcal{Q}=\left\{\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{P}_{i}, \boldsymbol{K}\right\}$ and consider the displacements of a point $\boldsymbol{P}_{\boldsymbol{O}}$ along the diagonal $\boldsymbol{P} \boldsymbol{P}_{\boldsymbol{i}}$, that is, $\boldsymbol{P}_{\boldsymbol{O}}(m)=$ $\boldsymbol{P}+m \cdot \boldsymbol{P} \boldsymbol{P}_{i}$. Then the function $D_{\text {Oddy }}(m)=D_{\text {Oddy }}(\mathcal{Q}(m))$, where
$\mathcal{Q}(m)=\left\{\boldsymbol{P}_{\boldsymbol{O}}(m), \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}$.
is the quadrilateral with vertices $\boldsymbol{P}_{\boldsymbol{O}}(m), \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}$ in counter-clockwise order that has a global minimum.

Proof. See Appendix B.

(a) Oddy distortion in the mesh before smoothing. The greater the distortion, the darker the shading (average distortion $=0.31 ; 99$ th percentile distortion=4.27).

(b) Relative side size error in mesh before smoothing. The greater the error, the darker the shading (average relative error $=8.69 \%$; the $68 \%$ of edges have a relative error lower or equal than $10 \%$ ).

Fig. 9. Quality measurements in the original mesh.

From the proof of Proposition 2.2 we can explicit compute the absolute minimum value $m^{*}$ for the Oddy distortion measure. Thus substituting the value $m^{*}$ in $\boldsymbol{P}_{\boldsymbol{O}}(m)$ we a obtain a vertex $\boldsymbol{P}_{\boldsymbol{O}}\left(m^{*}\right)$ and conclude that the length $L_{i}=\left\|\boldsymbol{P}_{\boldsymbol{O}}\left(\boldsymbol{m}^{*}\right) \boldsymbol{P}_{i}\right\|$ is associated to a quadrilateral $\mathcal{Q}^{*}:=\left\{\boldsymbol{P}_{\boldsymbol{O}}\left(m^{*}\right), \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}$ having an diagonal with an optimal length for the set of quadrilaterals $\mathcal{Q}(m)=\left\{\boldsymbol{P}_{\boldsymbol{O}}(m), \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}$ that are obtained by displacing $\boldsymbol{P}$ along the diagonal $\boldsymbol{P} \boldsymbol{P}_{\boldsymbol{i}}$.

Next, we will define an elasticity modulus $E_{i}(t)$ for a quadrilateral diagonal by using of the Oddy distortion measure.

### 2.2.3. The modulus of elasticity $E_{i}$ for a quadrilateral diagonal

The non-linear strain-stress behaviour of diagonals is implicitly introduced by formulating their $E_{i}(t)$ modulus as a function of the Oddy distortion, turning it unnecessary to adopt an exponential function as we do for the quadrilateral sides, as follows. In a similar way as above,


Fig. 10. Error of the size variation in structured mesh areas, before smoothing, expressed as growth slope error in degrees. The greater the error, the darker the shading (maximum $=18.98^{\circ}$; average of absolute value $=0.60^{\circ}$ ).
let $\mathcal{Q}=\left\{\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}$ be a quadrilateral. Now, let $\boldsymbol{P}_{\boldsymbol{S}}(\mathbf{t}):=\boldsymbol{P}+\mathbf{t}$ and consider the parametrized quadrilaterals $\mathcal{Q}(\mathbf{t}):=\left\{\boldsymbol{P}_{\boldsymbol{S}}(\mathbf{t}), \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}$, where $\mathcal{Q}(\mathbf{0})=\mathcal{Q}$. Then we define the modulus of elasticity for a quadrilateral diagonal as

$$
\begin{equation*}
E_{i}(\boldsymbol{t})=1+m_{d} \cdot D_{\text {Oddy }}(\mathcal{Q}(\mathbf{t})) \tag{16}
\end{equation*}
$$

where $m_{d}$ is a scalar to equilibrate the compromise between element size quality and distortion quality. The greater $m_{d}$, the more weight the diagonals will have in the final result, and vice versa. In our experience, $m_{d}=0.5$ provides a satisfactory compromise and optimal results. Taking smaller $m_{d}$-values reduce deviations from the desired element size at the cost of increasing the distortion. On the other hand, taking greater $m_{d}$-values can slightly reduce distortion at the cost of increasing the element size error. Although the diagonals are mainly responsible for the distortion variation, they also have an impact in the variations of the quadrilateral size. Thus, it is convenient to scale their goal length $L_{i}$ by a factor equal to the ratio between the desired quadrilateral size and the actual quadrilateral size. In this way, diagonals not only reduce distortion, but also help the sides to reach the desired quadrilateral size. We must also take into account that we are following the original formulation for the Oddy distortion, as previously stated. Thus, we must consider the sign of the area of the local parallelogram when evaluating the distortion at each vertex in the quadrilateral. In other words, if we obtain a null or negative area while computing (14), we shall give an arbitrarily large distortion value to that vertex. That way, degenerate quadrilaterals will have a large $E_{i}(t)$ modulus for its diagonals, due to the great need of such diagonals achieving their goal length, and thus fixing the quadrilateral degeneracy. As we are detecting the cases with null or negative area, we avoid accepting the distortion of degenerate quadrilaterals, as well as we make it impossible for asymptotes to appear from division by a null area. Since quadrilateral diagonals have a different modulus of elasticity that its corresponding sizes, we need to study $D_{\text {Oddy }}(\mathcal{Q}(t))$ as a function of $t=\left(t^{x}, t^{y}\right)$ in order to understand the nature of the function
$f_{P}^{(i)}(t):=\frac{v_{i}+t}{\left\|v_{i}+t\right\|} \cdot \frac{\left(\left\|v_{i}+t\right\|-L_{i}\right) E_{i}(t)}{L_{i}}=g_{i}(t) E_{i}(t)$
that we can write as the product of a differentiable function $g_{i}(t)=$ $\frac{\boldsymbol{v}_{i}+\boldsymbol{t}}{\left\|\boldsymbol{v}_{i}+t\right\|} \cdot \frac{\left(\left\|\boldsymbol{v}_{i}+t\right\|-L_{i}\right)}{L_{i}}$ and function $E_{i}(t)$ for each $1 \leq i \leq n_{d}$.

To this end, assume that $\boldsymbol{P}_{\boldsymbol{S}}(\mathbf{t})=\left(P^{x}+t^{x}, P^{y}+t^{y}\right), \boldsymbol{J}=\left(\boldsymbol{J}^{x}, J^{y}\right), \boldsymbol{P}_{i}=$ $\left(P_{i}^{x}, P_{i}^{y}\right), K=\left(K^{x}, K^{y}\right)$. Then, to evaluate $D_{\text {Oddy }}(\mathcal{Q}(t))$ we will use the following four functions (see (14)):
(1) $D_{\text {Oddy }, P_{S}}\left(P_{i}^{x}, J^{x}, P^{x}+t^{x}, P_{i}^{y}, J^{y}, P^{x}+t^{y}\right)$,

(a) Oddy distortion after Giuliani smoothing. The greater the distortion, the darker the shading (average distortion $=0.16 ; 99$ th percentile distortion $=1.56)$.

(b) Oddy distortion after Giuliani smoothing with the modifications by [3]. The greater the distortion, the darker the shading (average distortion $=0.14 ; 99$ th percentile distortion $=1.65)$.

Fig. 11. Oddy distortion after smoothing methods that ignore element size.
(2) $D_{\mathrm{Oddy}, J}\left(K^{x}, P_{i}^{x}, J^{x}, K^{y}, P_{i}^{y}, J^{y}\right)$,
(3) $D_{\mathrm{Oddy}, \boldsymbol{P}_{i}}\left(P^{x}+t^{x}, K^{x}, P_{i}^{x}, P^{y}+t^{y}, K^{y}, P_{i}^{y}\right)$,
(4) $D_{\mathrm{Oddy}, \boldsymbol{K}}\left(J^{x}, P^{x}+t^{x}, K^{x}, J^{y}, P^{y}+t^{y}, K^{y}\right)$.

Thus, $D_{\text {Oddy }}\left(\mathcal{Q}\left(t^{x}, t^{y}\right)\right)$ is computed as the maximum value against the above four functions, where 1,3 and 4 depend on $\left(t^{x}, t^{y}\right)$ and 2 is a constant function. In consequence, $D_{\text {Oddy }}\left(\mathcal{Q}\left(t^{x}, t^{y}\right)\right)$ is a $C^{2}$ piecewise smooth function [17] and hence it is semi-smooth [18]. Since $D_{\text {Oddy }}(\mathcal{Q}(\mathbf{t}))$ is a semi-smooth function then we can conclude that $E_{i}(\mathbf{t})$ is also semi-smooth and hence $f_{P}^{(i)}(t)=g_{i}(t) E_{i}(t)$ is a semi-smooth function for each $1 \leq i \leq n_{d}$.

Remark 2.3. We introduced the Individual Smoothing Procedure (ISP) function $f_{\boldsymbol{P}}$ as a sum of two class of functions: $\boldsymbol{f}_{\boldsymbol{P}}^{(j)}$ that are $\mathcal{C}^{2}$-smooth functions for $1 \leq j \leq n_{s}$ and $\boldsymbol{f}_{\boldsymbol{P}}^{(k)}$, that are semi-smooth for $1 \leq k \leq n_{d}$.

(a) Relative side size error after Giuliani smoothing. The greater the error, the darker the shading (average relative error $=7.77 \%$; the $72 \%$ of edges have a relative error lower or equal than $10 \%$ ).

(b) Relative side size error after Giuliani smoothing with the modifications by [3]. The greater the error, the darker the shading (average relative error $=9.33 \%$; the $64 \%$ of edges have a relative error lower or equal than $10 \%$ ).

Fig. 12. Relative side size error after smoothing methods that ignore element size.

Thus, we can conclude that the Individual Smoothing Procedure (ISP) function $f_{P}$ is a semi-smooth function.

### 2.3. An elasticity-based algorithm

Now, we have all ingredients to propose the following:
Individual Smoothing Procedure (ISP): Given $\boldsymbol{P} \in \mathcal{V}(\mathcal{Q}(\Omega))$ construct $f_{P}$ and compute $t^{*}$ satisfying $f_{\boldsymbol{P}}\left(t^{*}\right)=\mathbf{0}$. Return $\boldsymbol{P}_{S}$.

It allows us to introduce the following two smoothing procedures for $\mathfrak{Q}(\Omega)$.

## Sequential Procedure:

(1) Put $\mathcal{V}_{N S}(\mathfrak{Q}(\Omega)) \leftarrow \mathcal{V}(\mathfrak{Q}(\Omega))$;
(2) Repeat until $\mathcal{V}_{N S}(\mathfrak{Q}(\Omega))=\emptyset$ :

(a) Error of the size variation in structured mesh areas after Giuliani smoothing, expressed as growth slope error in degrees. The greater the error, the darker the shading ( maximum $=3.75^{\circ}$; average of absolute value $=0.39^{\circ}$ ).

(b) Error of the size variation in structured mesh areas after Giuliani smoothing with the modifications by [3], expressed as growth slope error in degrees. The greater the error, the darker the shading (maximum $=4.97^{\circ}$; average of absolute value $\left.=0.72^{\circ}\right)$.

Fig. 13. Error of the size variation after smoothing methods that ignore element size.
(a) Take $\boldsymbol{P} \in \mathcal{V}_{N S}(\mathfrak{Q}(\Omega))$;
(b) Run (ISP) to compute $\boldsymbol{P}_{\boldsymbol{S}}$;
(c) Put $\mathcal{V}(\mathfrak{Q}(\Omega)) \leftarrow(\mathcal{V}(\mathfrak{Q}(\Omega)) \backslash\{\boldsymbol{P}\}) \cup\left\{\boldsymbol{P}_{S}\right\}$;
(d) Put $\mathcal{V}_{N S}(\mathfrak{Q}(\Omega)) \leftarrow \mathcal{V}_{N S}(\mathfrak{Q}(\Omega)) \backslash\{\mathbf{P}\}$;
(3) Return $\mathcal{V}(\mathfrak{Q}(\Omega))$.

Let $D=\{1,2, \ldots, d\}$ be the number of available computational devices to implement (SP) in parallel.

## Parallel Procedure :

(1) Put $\mathcal{V}_{N S}(\mathfrak{Q}(\Omega)) \leftarrow \mathcal{V}(\mathfrak{Q}(\Omega))$ and $\mathcal{V}_{S}(\mathfrak{Q}(\Omega)) \leftarrow \emptyset$;
(2) Repeat until $\mathcal{V}_{N S}(\mathfrak{Q}(\Omega))=\emptyset$ :

(a) Oddy distortion after smoothing by minimization of the product of the area size error by the distortion average. The greater the distortion, the darker the shading (average distortion $=0.18 ; 99$ th percentile distortion $=0.90$ ).

(b) Oddy distortion after smoothing by the proposed algorithm. The greater the distortion, the darker the shading (average distortion $=0.15 ; 99$ th percentile distortion $=1.04$ ).

Fig. 14. Oddy distortion after smoothing methods that take care of element size.
(a) Take $\boldsymbol{P}^{(1)}, \ldots, \boldsymbol{P}^{(d)} \in \mathcal{V}_{N S}(\mathfrak{Q}(\Omega))$;
(b) Run (ISP) in device $1 \leq i \leq d$ to compute $\boldsymbol{P}_{S}^{(i)}$;
(c) Put $\mathcal{V}_{S}(\mathfrak{Q}(\Omega)) \leftarrow \mathcal{V}_{S}(\mathfrak{Q}(\Omega)) \cup\left\{\boldsymbol{P}^{(1)}, \ldots, \boldsymbol{P}^{(d)}\right\}$;
(d) $\mathcal{V}_{N S}(\mathfrak{Q}(\Omega)) \leftarrow \mathcal{V}_{N S}(\mathfrak{Q}(\Omega)) \backslash\left\{\boldsymbol{P}^{(1)}, \ldots, \boldsymbol{P}^{(d)}\right\}$;
(3) Return $\mathcal{V}_{S}(\mathfrak{Q}(\Omega))$.

The difference between the above two procedures is 2.(c). Recall that the set $\mathcal{V}(\mathfrak{Q}(\Omega))$ is used to compute $\mathcal{C}(\boldsymbol{P})$ that allows to construct $f_{\boldsymbol{P}}$. In the Sequential procedures.(c) we modify the set $\mathcal{V}(\mathfrak{Q}(\Omega))$ leaving the vertex $\boldsymbol{P}$ and introducing the smoothed one $\boldsymbol{P}_{S}$. Thus, there are vertices $\boldsymbol{P}^{\prime} \in \mathcal{V}(\mathfrak{Q}(\Omega))$ such that before we run (SP) it holds $\boldsymbol{P}^{\prime} \boldsymbol{P} \in$ $\mathcal{C}\left(\boldsymbol{P}^{\prime}\right)$, however after running (SP) we have $\boldsymbol{P}^{\prime} \boldsymbol{P} \notin \mathcal{C}\left(\boldsymbol{P}^{\prime}\right)$ (because in the Sequential Procedure 2.(c) we leave $\boldsymbol{P}$ from $\mathcal{V}(\mathfrak{Q}(\Omega))$. On the other hand, in the Parallel Procedure the set $\mathcal{V}(\mathfrak{Q}(\Omega))$ remains invariant and,

(a) Relative side size error after smoothing by minimization of the product of the area size error by the distortion average. The greater the error, the darker the shading (average relative error $=8.10 \%$; the $69 \%$ of edges have a relative error lower or equal than $10 \%$ ).

(b) Relative side size error after smoothing by the proposed algorithm. The greater the error, the darker the shading (average relative error $=7.35 \%$; the $75 \%$ of edges have a relative error lower or equal than $10 \%$ ).

Fig. 15. Relative side size error after smoothing methods that take care of element size.
in consequence, we compute each $\boldsymbol{P}_{S}$ in an independent way allowing to parallelize the algorithm.

## 3. On the practical implementation of the elasticity-based smooth-

 ing algorithmAfter having presented the values and conditions upon which the proposed smoothing algorithm is based, in this section we proceed to describe how it was implemented.

The smoothing is performed with an iterative approach. At each iteration, vertices interior to the mesh are moved to their smoothed position. The procedure ends when the vertices displacements are below a

(a) Error of the size variation in structured mesh areas after smoothing by minimization of the product of the area size error by the distortion average, expressed as growth slope error in degrees. The greater the error, the darker the shading (maximum $=5.30^{\circ}$; average of absolute value $=0.92^{\circ}$ ).

(b) Error of the size variation in structured mesh areas
after smoothing by the proposed algorithm, expressed as growth slope error in degrees. The greater the error, the darker the shading (maximum $=2.65^{\circ}$; average of absolute value $=0.37^{\circ}$ ).

Fig. 16. Error of size variation after smoothing methods that take care of element size.
given threshold in an iteration. A high quality is often already achieved within the first iterations, which permits to perform a partial smoothing from a fixed and low number of iterations. However, the quality of the elements' size gets improved as more iterations are completed. Therefore, by iterating until the vertices displacement threshold is met, we can reach optimum quality. It is possible to displace all vertices simultaneously at the end of each iteration (thus allowing a parallel implementation) or, alternatively, for sequential implementations only, each vertex can be displaced immediately once its equilibrium is solved (this makes the rest of vertices in the same iteration to benefit from considering previously solved vertices in their smoothed position, helping reduce the number of needed iterations). The achieved quality is almost
identical in the two approaches, although the result can be different (displacing all the vertices in parallel maintains symmetry if the mesh is symmetrical, while displacing vertices one by one as they are solved can lead to lose the symmetry in some cases). At each iteration, we locate all the springs that are connected to each vertex. For example, in Fig. 8, vertex A is shared by 5 quadrilaterals, and so we must consider its equilibrium from 10 springs ( 5 springs are quadrilateral sides, and the other 5 springs are diagonals connected to the vertex).

The equilibrium at the vertex is equation (6), where $f(t)$ is generated by adding the contribution of all springs connected to the vertex (both the ones belonging to sides and the ones from diagonals, expression (7)), with the only difference that $E_{i}(t)$ is taken from (10)


Fig. 17. Oddy distortion in a perforated panel (3872 quads)..
for quadrilateral sides, and from (16) for diagonals, and also with $L_{i}$ being the user-specified desired element size in the case of sides, or the ideal length in the case of diagonals (obtained from the best candidate in B.16). From Eq. (6) we get the $t$ value that transforms the vertex into its smoothed position. This equation can be solved by NewtonRaphson, building the Jacobian matrix as the sum as the contribution from sides springs - expression (11)- and from diagonals springs expression (17). Proceeding in this way for all vertices, we complete an iteration, and we continue until all $t$ vectors for all vertices are below a threshold. Pseudocode for the method is shown at algorithm 1 for
the sequential version in which each vertex is displaced as soon as its equilibrium is solved, and at algorithm 2 for the parallel version that displaces all the vertices simultaneously at the end of each iteration.

Now, in order to implement the proposed elasticity-based smoothing algorithm we need to compute for each $P \in \mathcal{V}(\mathfrak{Q}(\Omega))$ the value $t^{*}$ for which $f_{P}\left(\boldsymbol{t}^{*}\right)=\mathbf{0}$. Since the map $f_{P}$ is a semi-smooth function then, in order to use the Newton method to compute $t^{*}$, we need to substitute the classical derivative by a generalized one. More precisely, we use the so-called B-subdifferential ("B" for Bouligard) and the Clarke's generalized Jacobian (see Section 2.1 in [18]) that are defined below.


Fig. 18. Mesh before smoothing. Cross section of the Cathedral of Mallorca through the second arch [19] (10840 quads)..

Definition 3.1. Let $V \subset \mathbb{R}^{n}$ be an open set and $f: V \longrightarrow \mathbb{R}^{m}$ be Lipschitz continuous near $x \in V$. The set
$\partial_{B} f(x)=\left\{M \in \mathbb{R}^{n \times m}:\right.$ exists $\left(\boldsymbol{x}_{k}\right) \subset \operatorname{Dom}(\boldsymbol{f}): \boldsymbol{x}_{k} \rightarrow \boldsymbol{x}$ and $\left.D \boldsymbol{f}\left(\boldsymbol{x}_{k}\right) \rightarrow M\right\}$
is called the B-subdifferential of $f$ at $x$. Moreover, Clarke's generalized Jacobian of $f$ at $x$ is the convex hull of the B-subdifferential of $f$ at $x$, that is,
$\partial f(x)=\operatorname{co}\left(\partial_{B} f(x)\right)$.

It is possible to prove that if $f$ is continuously differentiable in a neighbourhood of $x$ then
$\partial f(x)=\partial_{B} f(x)=\{D f(x)\}$
(see Proposition 2.2 (e) in [18]).
Moreover, in the case of piecewise differentiable maps it can be shown the following result (see Proposition 4.3.1 in [17]).

Proposition 3.2. Let $V$ be an open set in $\mathbb{R}^{n}$ and $f: V \longrightarrow \mathbb{R}^{m}$ is a piecewise $\mathcal{C}^{1}$-function with $\mathcal{C}^{1}$-selection functions $\boldsymbol{f}_{i}: \mathcal{O} \longrightarrow \mathbb{R}^{m} 1 \leq i \leq k$

(a) Oddy distortion (mean $=0.13 ; 99$ th percentile $=2.08$ ).

(b) Error of the size variation in structured mesh areas $\left(\right.$ maximum $=6.13^{\circ} ;$ mean $\left.=0.26^{\circ}\right)$.

Fig. 19. Giuliani smoothing with the modifications by [3], applied to the cross section of the Cathedral of Mallorca through the second arch [19] (10840 quads)..
at $x_{0} \in \mathcal{O} \subset V$, that is, $\boldsymbol{f}(\boldsymbol{x}) \in\left\{\boldsymbol{f}_{1}(\boldsymbol{x}), \ldots, \boldsymbol{f}_{k}(\boldsymbol{x})\right\}$ for all $\boldsymbol{x} \in U$. Then $\partial f\left(x_{0}\right)=\operatorname{co}\left\{D f_{i}\left(x_{0}\right): i \in I_{f}\left(x_{0}\right)\right\}$,
where
$I_{\boldsymbol{f}}\left(\boldsymbol{x}_{0}\right)=\left\{1 \leq i \leq k: \boldsymbol{x}_{0} \in \operatorname{cl}\left(\operatorname{int}\left\{\boldsymbol{x} \in \mathcal{O}: \boldsymbol{f}(\boldsymbol{x})=\boldsymbol{f}_{i}(\boldsymbol{x})\right\}\right)\right\}$.
We have also available some basic calculus rules (see 2.3 in [20]).

Proposition 3.3. Let $V$ be an open set in $\mathbb{R}^{n}$ and $f, g: V \longrightarrow \mathbb{R}$ Assume that $f$ and $g$ are Lipschitz near $x \in V$. Then the following statements hold.
(a) $\partial(s f)(x)=s \partial f(x)$ for any scalar $s$.
(b) $\partial(f+g)(x) \subset \partial f(x)+\partial g(x)$ and if either $f$ or $g$ is differentiable at $\boldsymbol{x}$ then $\partial(f+g)(x)=\partial f(\boldsymbol{x})+\partial g(\boldsymbol{x})$ holds.
(c) The map $(f \cdot g)(x)=f(x) g(x)$ is Lipschitz near $\boldsymbol{x} \in V$ and $\partial(f \cdot g)(\boldsymbol{x}) \subset f(\boldsymbol{x}) \partial g(\boldsymbol{x})+g(\boldsymbol{x}) \partial f(\boldsymbol{x})$.
(d) If $g(x) \neq 0$ then the map $\left(\frac{f}{g}\right)(x)=\frac{f(x)}{g(x)}$ is Lipschitz near $x \in V$ and

$$
\partial\left(\frac{f}{g}\right)(\boldsymbol{x}) \subset \frac{g(\boldsymbol{x}) \partial f(\boldsymbol{x})-f(\boldsymbol{x}) \partial g(\boldsymbol{x})}{g^{2}(\boldsymbol{x})}
$$

[^1]
(a) Oddy distortion (mean $=0.14 ; 99$ th percentile $=1.71$ ).

(b) Error of the size variation in structured mesh areas $\left(\right.$ maximum $=6.07^{\circ} ;$ mean $\left.=0.26^{\circ}\right)$.
 arch [19] (10840 quads)..

Corollary 3.4. The Individual Smoothing Procedure (ISP) function $f_{P}$ satisfies
$\partial \boldsymbol{f}_{\boldsymbol{P}}(t)=\left\{D \boldsymbol{f}_{\boldsymbol{P}}^{(s)}(t)\right\}+\partial \boldsymbol{f}_{\boldsymbol{P}}^{(d)}(t)$.
Moreover, since $f_{P}^{(d)}(t)=\sum_{k=1}^{n_{d}} g_{i}(t) E_{i}(t)$ we have
$\partial f_{P}^{(d)}(t) \subset \sum_{k=1}^{n_{d}} E_{i}(t)\left\{D g_{i}(t)\right\}+g_{i}(t) \partial E_{i}(t)$
This corollary allows to introduce the semi-smooth Newton method to compute $\mathrm{t}^{*}$ such that $\boldsymbol{f}_{P}\left(\boldsymbol{t}^{*}\right)=\mathbf{0}$ holds.

## Semi-smooth Newton Method

1. Choose an initial $t_{0}$ and set $k=0$.
2. If $\boldsymbol{f}_{\boldsymbol{P}}\left(\boldsymbol{t}_{\boldsymbol{k}}\right)=\mathbf{0}$ then STOP.
3. Choose $M_{k} \in \partial f_{P}^{(d)}\left(t_{k}\right)$ and compute $s_{k}$ from $\left(D f_{P}^{(s)}\left(t_{k}\right)+M_{k}\right) s_{k}=$ $-f_{P}\left(t_{k}\right)$.
4. Set $t_{k+1}=t_{k}+s_{k}$ and goto to step 2 .

In our practical implementation of the semi-smooth Newton method we use a matrix $M_{k}$ constructed as follows. Assuming that $E_{i}(t)$ is a differentiable function of $t$ then the Jacobian matrix $D f_{P}^{(d)}(t)$ is

(a) Oddy distortion (mean $=0.13 ; 99$ th percentile $=1.40$ ).

(b) Error of the size variation in structured mesh areas $\left(\right.$ maximum $=3.45^{\circ} ;$ mean $\left.=0.16^{\circ}\right)$.

Fig. 21. Our smoothing algorithm applied to the cross section of the Cathedral of Mallorca through the second arch [19] (10840 quads)..
constructed by using
$D f_{P}^{(d)}(t)=\sum_{k=1}^{n_{d}} E_{i}(t) D g_{i}(t)+g_{i}(t) D E_{i}(t)$
where
$D E_{i}(t)=\left(\begin{array}{cc}\frac{\partial E_{i}(t)}{\partial t^{x}} & \frac{\partial E_{i}(t)}{\partial t^{y}}\end{array}\right)$
Then we use $M_{k}:=M\left(\boldsymbol{t}_{\boldsymbol{k}}\right)$ for a function $M$ defined by
$M(t):=\sum_{k=1}^{n_{d}} E_{i}(t) D g_{i}(t)+g_{i}(t) \delta D_{\text {Oddy }}(t) t^{T}$,
where $\delta D_{\text {Oddy }}(\boldsymbol{t})$ is defined as follows. From Proposition 2.2 we know the existence of a minimum value $D_{\text {Oddy }}\left(\mathcal{Q}\left(\boldsymbol{m}^{*}\right)\right)$ associated to the diagonal $\boldsymbol{P}_{\boldsymbol{O}}\left(\boldsymbol{m}^{*}\right) \boldsymbol{P}_{\boldsymbol{i}}$ used to compute the value $L_{i}$, in the diagonal case. Now, we consider the quadrilateral
$\mathcal{Q}(t)=\left\{\boldsymbol{P}_{S}(\boldsymbol{t}), \boldsymbol{J}, \boldsymbol{P}_{i}, \boldsymbol{K}\right\}$,
recall that $\boldsymbol{P}_{\boldsymbol{S}}(\boldsymbol{t})=\boldsymbol{P}+\boldsymbol{t}$, then we define
$\delta D_{\text {Oddy }}(t):=\frac{D_{\text {Oddy }}\left(\mathcal{Q}\left(\boldsymbol{m}^{*}\right)\right)-D_{\text {Oddy }}(\mathcal{Q}(\boldsymbol{t}))}{\left\|\boldsymbol{P}_{\boldsymbol{O}}\left(\boldsymbol{m}^{*}\right) \boldsymbol{P}_{\boldsymbol{S}}(\boldsymbol{t})\right\|}$,

(a) Oddy distortion (mean $=0.64 ; 99$ th percentile $=7.17$ ).

(b) Error of the size variation in structured mesh areas $\left(\right.$ maximum $=21.79^{\circ} ;$ mean $\left.=0.86^{\circ}\right)$.

Fig. 22. Mesh before smoothing. Bicycle chainring (9935 quads)..
by using that $\boldsymbol{P}_{\boldsymbol{O}}\left(\boldsymbol{m}^{*}\right)$ provides a minimum distortion, and this let us use $\delta D_{\text {Oddy }}(t)$ as a measure of the distortion variation in $t$. Our implementation uses the following semi-smooth method:

## Implemented semi-smooth Newton Method

1. Choose an initial $t_{0}$ and set $k=0$.
2. If $\boldsymbol{f}_{\boldsymbol{P}}\left(\boldsymbol{t}_{\boldsymbol{k}}\right)=\mathbf{0}$ then STOP.
3. Compute $s_{k}$ from $\left(D f_{P}^{(s)}\left(t_{k}\right)+M\left(t_{k}\right)\right) s_{k}=-f_{P}\left(t_{k}\right)$.
4. Set $t_{k+1}=t_{k}+s_{k}$ and goto to step 2 .

(a) Oddy distortion (mean $=0.37$; 99th percentile $=3.57$ ).

(b) Error of the size variation in structured mesh areas (maximum $=11.15^{\circ} ;$ mean $=0.96^{\circ}$ ).

Fig. 23. Giuliani smoothing with the modifications by [3], applied to a bicycle chainring (9935 quads)..

### 3.1. A comparative analysis from different smoothing algorithms

We show a comparative analysis of the results obtained from different smoothing algorithms. A set of examples can be seen in Figures Figs. 9 to 16.

We have measured the Oddy distortion, the side size error (relative in percentage respect of the desired size), and the growth error (this last one in areas with structured meshing only, as it is truly meaningful in consecutive collinear edges - which occur mainly when there are


Fig. 24. Smoothing by minimization of the product of the area size error by the distortion average applied to a bicycle chainring ( 9935 quads)..
four edges per vertex, that is, structured meshing). The growth error has been measured by comparing the ratio between the lengths of two consecutive collinear edges, and the ratio that they should have if their lengths were the desired ones. The result is expressed as the growth angle error, in degrees. Table 1 shows all these quality measurements, for the original mesh without smoothing (Figs. 9 and 10), the original Giuliani algorithm [6] (Figs. 11(a), 12(a), and 13(a)), the Giuliani algorithm as modified by Sarrate and Huerta [3] (Figs. 11(b), 12(b), and $13(\mathrm{~b})$ ), the method with the distortion by size error product minimization (reinterpreting Gargallo, Roca and Sarrate [7] with the Oddy distortion, Figs. 14(a), 15(a), and 16(a)), as well as our proposed
spring-based algorithm (Figs. 14(b), 15(b), and 16(b)). The results show that the proposed algorithm provides an optimal response to our requirements. Regarding Oddy distortion, the obtained quality is better than that of the Giuliani-based algorithms (the average is similar, but the 99th percentile of the distortion gets reduced in about $40 \%$ - an important aspect, as those other algorithms should achieve an extraordinarily low distortion given that they do not take care of meeting the user desired element size). The distortion from the proposed method is very similar to the one provided by the product minimization algorithm (the average is reduced from 0.18 to 0.15 , while the 99th percentile is slightly increased from 0.90 to 1.04). At

Table 1
Comparative analysis of quality measures after each studied algorithm. The proposed algorithm is optimal, showing a remarkable reduction at the growth error (higher fidelity to size gradients required by the user). It also reduces the size error from the other algorithms, and achieves an optimal Oddy distortion (significantly better than the Giuliani-based methods, and on a par with the distortion-size product minimization method).
$\left.\begin{array}{lllllll}\hline & \begin{array}{l}\text { Average } \\ \text { distortion }\end{array} & \begin{array}{l}\text { Distortion 99th } \\ \text { percentile }\end{array} & \begin{array}{l}\text { Average size } \\ \text { error }\end{array} & \begin{array}{l}\text { Sides with } \\ \text { error } \leqslant 10 \%\end{array} & \begin{array}{l}\text { Maximum } \\ \text { growth error }\end{array} \\ \hline \text { of growth error }\end{array}\right]$

```
Version displacing each vertex as soon as its equilibrium is solved
input: {vertex list},
        {quadrilateral list},
        {desired element size (per vertex)}
repeat
    foreach P}\leftarrow\mathrm{ interior vertex in {vertex list}
        t}\leftarrow[0,0
        repeat
            f}\leftarrow[0,0
            J}\leftarrow[0,0,0,0
            foreach A}\leftarrowq\mathrm{ quadrilateral side connected to P
                Li}\leftarrow\mathrm{ desired A length (from user input)
            f}\leftarrow\textrm{f}+\textrm{f}(\textrm{A},\mp@subsup{L}{i}{},\textrm{t}) -evaluating equation (7)
            J}\leftarrow\textrm{J}+\textrm{J}(\textrm{A},\mp@subsup{L}{i}{},\textrm{t}) -evaluating equation (11
            foreach D & quadrilateral diagonal connected to P
                m}\leftarrow\mathrm{ best }\mp@subsup{m}{i}{}\mathrm{ candidate from B. 16
                Li}\leftarrow\mp@code{Ideal length for D according to scalar m
                (optional) }\mp@subsup{L}{i}{}\leftarrow\mp@subsup{L}{i}{}\cdot\mathrm{ mean( }\mp@subsup{L}{i}{}\mathrm{ of sides)/mean(side lengths)
                f}\leftarrow\textrm{f}+\textrm{f}(\textrm{D},\mp@subsup{L}{i}{},\textrm{t}) -evaluating equation (7) with (16)
                J }\leftarrow\textrm{J}+\textrm{J}(\textrm{D},\mp@subsup{L}{i}{},\textrm{t}) -evaluating equation (17)
            tnew}\leftarrow\mathrm{ solve the system J(t).(t tew - t) = -f(t)
            step }\leftarrow\mp@subsup{t}{\mathrm{ new }}{}-\textrm{t
            t}\leftarrow\mp@subsup{t}{\mathrm{ new}}{
        while (norm(f) > threshold (sol ) AND (norm(step) > threshold step)
        P}\leftarrowP+
while max( norm(t) ) > threshold smoothing
```

Listing 1: Proposed smoothing algorithm (sequential).
the edge size error, our algorithm gets the best result of all the studied methods, with an average relative error of $7.35 \%$, and managing that $75 \%$ of edges have a relative error below $10 \%$. The other methods do not achieve these results. The low value of the growth error is also noteworthy, being about the half of the maximum error from the other algorithms (let us recall that the example used in the comparison has a size gradient required by the user, and therefore the growth error is also an important quality measurement). With a maximum error of $2.65^{\circ}$ and with $0.37^{\circ}$ as the average of the absolute value of the error, it is the algorithm that best maintains desired size gradients. A quick glimpse at Table 1 could give the false impression that the original Giuliani algorithm provides a quality not too far away from our proposed method, even if its results are worse ${ }^{1}$. However, as said before, it tends to make all elements of uniform size, and this does not substantially affect the quality measurements but can be visually appreciated at Fig. 11(a) (the size gradient desired by the user has lost its presence to a large degree). In conclusion, it can be observed that the proposed algorithm provides an excellent response to the required quality criteria, as we wished to decrease as much as possible the size error, respecting the size gradients desired by the user, and with an as

[^2]```
Version with simultaneous displacement of all vertices
input: {vertex list},
    {quadrilateral list},
        {desired element size (per vertex)}
Ps }\leftarrow{\mathrm{ smoothed vertex list}}\leftarrow{vertex list
repeat
    foreach P }\leftarrow\mathrm{ interior vertex in {vertex list}
        t}\leftarrow[0,0
            repeat
            f}\leftarrow[0,0
            J}\leftarrow[0,0,0,0
            foreach A}\leftarrowq\mathrm{ quadrilateral side connected to P
                Li}\leftarrow\mp@code{desired A length (from user input)
                f}\leftarrow\textrm{f}+\textrm{f}(\textrm{A},\mp@subsup{L}{i}{},\textrm{t})-\mathrm{ -valuating equation (7)
                J}\leftarrow\textrm{J}+\textrm{J}(\textrm{A},\mp@subsup{L}{i}{},\textrm{t}) -evaluating equation (11)
            foreach D }\leftarrow\mathrm{ quadrilateral diagonal connected to P
                m}\leftarrow\mathrm{ best }\mp@subsup{m}{i}{}\mathrm{ candidate from (B.16)
                L
                (optional) }\mp@subsup{L}{i}{}\leftarrow\mp@subsup{L}{i}{}\cdot\mathrm{ mean( }\mp@subsup{L}{i}{}\mathrm{ of sides)/mean(side lengths)
                f}\leftarrow\textrm{f}+\textrm{f}(\textrm{D},\mp@subsup{L}{i}{},\textrm{t}) -evaluating equation (7) with (16)
                J}\leftarrow\textrm{J}+\textrm{J}(\textrm{D},\mp@subsup{L}{i}{},\textrm{t}) -evaluating equation (17)
            tnew}\leftarrow\mathrm{ solve the system J(t).(t tew - t) = -f(t)
            step }\leftarrow\mp@subsup{t}{\mathrm{ new }}{}-\textrm{t
            t}\leftarrow\mp@subsup{t}{\mathrm{ new}}{
            while (norm(f) > threshold dsol ) AND (norm(step) > threshold (step)
        Ps[P]}\leftarrow\textrm{P}+\textrm{t
    foreach P }\leftarrow\mathrm{ interior vertex in {vertex list}
        P}\leftarrow\textrm{Ps}[\textrm{P}
while max( norm(t) ) > threshold smoothing
```

Listing 2: Proposed smoothing algorithm (parallel).
low as possible quadrilateral distortion. The springs model, with the diagonals controlling the element distortion, and the edges taking care of the element size, achieves a great result meeting all these requirements, without neglecting any of them. Moreover, the algorithm also removes the "noise patterns" like those introduced by methods like Giuliani with the modifications from [3] (areas with elements sizes substantially different from the size they should have, creating visually noticeable patterns). By comparing Figs. 11(b) and 14(b), it can be observed how the proposed algorithm (second figure) has eliminated the presence of these patterns, which are noticeable in the first figure. In addition to quality, the algorithm can be implemented very efficiently both in $C P U$ and in GPU, thanks to its parallelization possibilities. In each iteration, the smoothing of each point can be made independent from the rest of points, which allows for a massive parallel implementation without communication between processing cores (communication is only necessary at the end of each iteration).

Figs. 17 to 25 show the algorithm applied to several models, where the quality of the results has been compared to other algorithms mentioned throughout the paper. The number of quadrilaterals in each

(a) Oddy distortion (mean $=0.33$; 99th percentile $=2.33$ ).

(b) Error of the size variation in structured mesh areas $\left(\right.$ maximum $=10.78^{\circ} ;$ mean $\left.=0.58^{\circ}\right)$.

Fig. 25. Our smoothing algorithm applied to a bicycle chainring (9935 quads)..
of the models has been chosen so that it is larger than the example that we have been discussing, while at the same time the mesh can still be appreciated.

As it can be seen across this set of figures, our approach achieves greater quality than other algorithms. Note that we designed our algorithm not to reduce distortion only, but to make the size of the elements as close as possible to their desired size, and to do so in a way that the desired size gradients are kept in the places where they should happen and with the slope they must have (so that gradients are not blurred nor moved away from their desired location). For this reason, in these
figures it is important to compare not just the distortion, but the size gradients as well.

It can also be seen that our choice of using [3] as the meshing algorithm shows a good behaviour in models that were not specifically tuned for being meshed. This is particularly noticeable not only around curved borders, but also in situations when the contour allows for a regular structured mesh, like the cross section of the Cathedral of Mallorca through the second arch.

## 4. Conclusions and future work

The proposed algorithm for smoothing post-processing, based on a spring system, provided optimal results in our experiments, being the one delivering the highest quality measurements from all the alternatives that we tested. It is suitable for both sequential and parallel execution, and it is easy to implement. We based the formulation on the Oddy distortion, but the development of the algorithm allows for adaptations to other distortion criteria, by just modifying the ideal length expressions for the quadrilateral diagonals accordingly (expressions (B.1) to B.16). The simplicity of its parallel implementation (2) suggests promising future work in the area of multi-core CPU and GPU programming, and our day-to-day use of the algorithm in our mesh-based simulations may open opportunities for improvement.

## Declaration of competing interest

The authors declare that they have no known competing financial interests or personal relationships that could have appeared to influence the work reported in this paper.

## Data availability

No data was used for the research described in the article.

## Acknowledgements

This research was funded by the RTI2018-093521-B-C32 grant from the Ministerio de Ciencia e Innovación, Spain and by the grant number INDI22/15 from Universidad CEU Cardenal Herrera, Spain.

## Appendix A. Proof of Lemma 2.1

$\underset{g_{P}^{(i)}(t)=\left(\boldsymbol{v}_{i}+t\right)}{\substack{\text { Recall that }\\}} \cdot \frac{\left(\left\|\boldsymbol{v}_{i}+t\right\|-L_{i}\right)}{L_{i}\left\|\boldsymbol{v}_{i}+t\right\|}$,
then

$$
\begin{aligned}
D g_{P}^{(i)}(t) & =\frac{\left(\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|-L_{i}\right)}{L_{i}\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|} D\left(\left(\boldsymbol{v}_{i}+\boldsymbol{t}\right)\right)+\left(\boldsymbol{v}_{i}+\boldsymbol{t}\right) D\left(\frac{\left(\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|-L_{i}\right)}{L_{i}\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|}\right) \\
& =\frac{\left(\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|-L_{i}\right)}{L_{i}\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|}\left(\begin{array}{ll}
1 & 0 \\
0 & 1
\end{array}\right)+\frac{1}{L_{i}}\left(\boldsymbol{v}_{i}+\boldsymbol{t}\right) D\left(\frac{\left(\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|-L_{i}\right)}{\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|}\right) .
\end{aligned}
$$

Now,

$$
\begin{aligned}
D\left(\frac{\left(\left\|v_{i}+t\right\|-L_{i}\right)}{\left\|v_{i}+t\right\|}\right) & =\frac{\left\|\boldsymbol{v}_{i}+t\right\| D\left(\left\|\boldsymbol{v}_{i}+t\right\|\right)+\left(\left\|v_{i}+t\right\|-L_{i}\right) D\left(\left\|v_{i}+t\right\|\right)}{\left\|\boldsymbol{v}_{i}+t\right\|^{2}} \\
& =\frac{L_{i} D\left(\left\|v_{i}+t\right\|\right)}{\left\|v_{i}+t\right\|^{2}}
\end{aligned}
$$

To conclude the proof, we take into account that
$D\left(\| \boldsymbol{v}_{i}+\boldsymbol{t \|}\right)=D\left(\sqrt{\left(\boldsymbol{v}_{i}+t\right)^{T}\left(\boldsymbol{v}_{i}+\boldsymbol{t}\right)}\right)=\frac{\left(\boldsymbol{v}_{i}+\boldsymbol{t}\right)^{T}}{\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|}$,
hence
$D g_{P}^{(i)}(t)=\frac{\left(\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|-L_{i}\right)}{L_{i}\left\|\boldsymbol{v}_{i}+t\right\|}\left(\begin{array}{ll}1 & 0 \\ 0 & 1\end{array}\right)+\left(\boldsymbol{v}_{i}+\boldsymbol{t}\right) \frac{\left(\boldsymbol{v}_{i}+\boldsymbol{t}\right)^{T}}{\left\|\boldsymbol{v}_{i}+\boldsymbol{t}\right\|^{3}}$.
This proves the lemma.

 the intersection between curves $\boldsymbol{J}$ and $\boldsymbol{K}$. The horizontal axis represents the $m$ scalar parametrizing $\boldsymbol{P}_{\boldsymbol{O}}(m)$.

## Appendix B. Proof of Proposition 2.2

Recall that, according to Eq. (15), the Oddy distortion will be the greatest of the local distortions at each of the four vertices in the quadrilateral. Since $\boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}$ remain in a fixed position, the Oddy distortion at the $\boldsymbol{P}_{\boldsymbol{i}}$ vertex remains constant, because the angle between the $\boldsymbol{J} \boldsymbol{P}_{\boldsymbol{i}}$ and the $\boldsymbol{P}_{\boldsymbol{i} \boldsymbol{K}}$ sides is kept constant. Therefore, to find the goal length for the $\boldsymbol{P}_{\boldsymbol{O}}(\boldsymbol{m}) \boldsymbol{P}_{\boldsymbol{i}}$-diagonal, we can ignore the Oddy distortion at $\boldsymbol{P}_{\boldsymbol{i}}$. Thus we only need to evaluate $D_{\text {Oddy }, i}(m):=D_{\text {Oddy }, i}\left(\left\{\boldsymbol{P}_{\boldsymbol{O}}(m), \boldsymbol{J}, \boldsymbol{P}_{\boldsymbol{i}}, \boldsymbol{K}\right\}\right)$ for $i \in\{\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{K}\}$. As we shall see, the variation of the Oddy distortion at each vertex in the set $\left\{\boldsymbol{P}_{\boldsymbol{O}}(m), \boldsymbol{J}, \boldsymbol{K}\right\}$ as we move $m$ yields in a cubic polynomial in $m$. Each cubic polynomial can have a feasible or an unfeasible minimum (because in some cases the minimum can correspond to an absurd geometry). Since the distortion is the maximum from the distortions evaluated at the vertices, we need to compute the intersection of the three cubic curves, in order to take into account the envelope of the maximum distortion.

In conclusion, the optimal position $\boldsymbol{P}_{\boldsymbol{O}}\left(m^{*}\right)$ will take place at exactly one of these points:
(i) A local minimum from one of the three curves.
(ii) An intersection point between them.

Fig. B. 26 shows an example in which the minimum Oddy distortion occurs at the intersection between the curves associated to vertices $\boldsymbol{J}$ and $\boldsymbol{K}$. This example illustrates the reason to find the local minima for the three curves and its intersection points in order to obtain the minimum distortion for the quadrilateral. To provide an explicit expression for $D_{\mathrm{Oddy}, i}(m)$ for $i \in\{\boldsymbol{P}, \boldsymbol{J}, \boldsymbol{K}\}$ we introduce the following notation.

$$
\begin{aligned}
& \text { - We put } \boldsymbol{P} \boldsymbol{P}_{i}=\left(P P_{i}^{x}, P P_{i}^{y}\right), \boldsymbol{P} \boldsymbol{J}=\left(P J^{x}, P J^{y}\right), \boldsymbol{P} \boldsymbol{K}=\left(P K^{x}, P K^{y}\right) \text {, } \\
& \boldsymbol{J} \boldsymbol{P}_{i}=\left(J P_{i}^{x}, J P_{i}^{y}\right), \boldsymbol{K} \boldsymbol{P}_{i}=\left(K P_{i}^{x}, K P_{i}^{y}\right), \boldsymbol{J} \boldsymbol{K}=\left(J K^{x}, J K^{y}\right), \boldsymbol{P} \boldsymbol{J}= \\
& \left(P J^{x}, P J^{y}\right) \text { and } \boldsymbol{P} \boldsymbol{K}=\left(P K^{x}, P K^{y}\right) .
\end{aligned}
$$

By using (14) together $\boldsymbol{P}_{\boldsymbol{O}}(m)$, we can evaluate the Oddy distortion at each vertex as a function of the $m$ or, in other words, as a function of the translation of $\boldsymbol{P}$ along the diagonal. We can arrive to the following expressions, which are the equations of the three curves plotted at Fig. B.26: (See Box I)
$D_{\text {Oddy }, J}(m)=2\left[\frac{\left[\left(P J^{x}-m P P_{i}^{x}\right)^{2}+\left(P J^{y}-m P P_{i}^{y}\right)^{2}+\left\|J P_{i}\right\|^{2}\right]^{2}}{4\left(-m P P_{i}^{x} J P_{i}^{y}+m P P_{i}^{y} J P_{i}^{x}-J P_{i}^{x} P J^{y}+J P_{i}^{y} P J^{x}\right)^{2}}-1\right]$,
$D_{\text {Oddy }, K}(m)=2\left[\frac{\left[\left(P K^{x}-m P P_{i}^{x}\right)^{2}+\left(P K^{y}-m P P_{i}^{y}\right)^{2}+\left\|\boldsymbol{K} \boldsymbol{P}_{i}\right\|^{2}\right]^{2}}{4\left(m P P_{i}^{x} K P_{i}^{y}-m P P_{i}^{y} K P_{i}^{x}+K P_{i}^{x} P K^{y}-K P_{i}^{y} P K^{x}\right)^{2}}-1\right]$.

## Computing the intersection points

In order to find the candidate values for $m$ that correspond to the intersection of the curves for the local distortion variation at $\boldsymbol{J}$ and $\boldsymbol{K}$, we need to solve the equation
$D_{\text {Oddy }, J}(m)=D_{\text {Oddy }, K}(m)$
which, after the substitution of expressions (B.2) y (B.3), implies finding the roots of a cubic equation in the form
$e_{J K} m^{3}+f_{J K} m^{2}+g_{J K} m+h_{J K}=0$
where
$e_{J K}=\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}(s-r)$
$f_{J K}=2(b r-a s)+\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}(t-u)$
$g_{J K}=2(b u-a t)-d r+c s$
$h_{J K}=c t-d u$
$a=P P_{i}^{x} P J^{x}+P P_{i}^{y} P J^{y}$
$b=P P_{i}^{x} P K^{x}+P P_{i}^{y} P K^{y}$
$c=\left\|\boldsymbol{J} \boldsymbol{P}_{i}\right\|^{2}+\|\boldsymbol{P} \boldsymbol{J}\|^{2}$
$d=\left\|\boldsymbol{K} \boldsymbol{P}_{i}\right\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}$
$r=P P_{i}^{y} J P_{i}^{x}-P P_{i}^{x} J P_{i}^{y}$
$s=P P_{i}^{x} K P_{i}^{y}-P P_{i}^{y} K P_{i}^{x}$
$t=K P_{i}^{x} P K^{y}-K P_{i}^{y} P K^{x}$
$u=J P_{i}^{y} P J^{x}-J P_{i}^{x} P J^{y}$

$$
\begin{equation*}
D_{\mathrm{Oddy}, P}(m)=2\left[\frac{\left[\left(P J^{x}-m P P_{i}^{x}\right)^{2}+\left(P K^{x}-m P P_{i}^{x}\right)^{2}+\left(P J^{y}-m P P_{i}^{y}\right)^{2}+\left(P K^{y}-m P P_{i}^{y}\right)^{2}\right]^{2}}{4\left[m\left[P P_{i}^{x}\left(P J^{y}-P K^{y}\right)+P P_{i}^{y}\left(P K^{x}-P J^{x}\right)\right]+P J^{x} P K^{y}-P J^{y} P K^{x}\right]^{2}}-1\right], \tag{B.1}
\end{equation*}
$$

Box I.
equation that can have three distinct real roots, three multiple real roots, or one real and two non-real roots.

Analogously, we can obtain the $m$ candidates belonging to the intersection between the $\boldsymbol{J}$ and $\boldsymbol{P}$ distortion variation curves, solving the equation
$D_{\text {Oddy }, J}(m)=D_{\text {Oddy }, P}(m)$
which leads to finding the roots of another cubic equation
$e_{J P} m^{3}+f_{J P} m^{2}+g_{J P} m+h_{J P}=0$
where
$e_{J P}=\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}(z-2 r)$
$f_{J P}=2(a r+b r-a z)+\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}(w-2 u)$
$g_{J P}=2(a u+b u-a w)+c z-r\left(\|\boldsymbol{P} \boldsymbol{J}\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}\right)$
$h_{J P}=c w-u\left(\|\boldsymbol{P} \boldsymbol{J}\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}\right)$
$w=P J^{x} P K^{y}-P J^{y} P K^{x}$
$z=P P_{i}^{y} J K^{x}-P P_{i}^{x} J K^{y}$
taking $a, b, c, r, u$ the same values as in Eq. (B.5). And we also add the $m$ candidates due to the intersection for the $\boldsymbol{K}$ and $\boldsymbol{P}$ curves:
$D_{\text {Oddy }, K}(m)=D_{\text {Oddy }, P}(m)$
resulting in the cubic equation
$e_{K P} m^{3}+f_{K P} m^{2}+g_{K P} m+h_{K P}=0$
where
$e_{K P}=\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}(z-2 s)$
$f_{K P}=2(a s+b s-b z)+\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}(w-2 t)$
$g_{K P}=2(a t+b t-b w)+d z-s\left(\|\boldsymbol{P} \boldsymbol{J}\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}\right)$
$h_{K P}=d w-t\left(\|\boldsymbol{P} \boldsymbol{J}\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}\right)$
taking $a, b, d, s, t, w, z$ the same values as in Eqs. (B.5) and (B.7). It can be observed that Eqs. (B.7) and (B.9) take symmetrical expressions against each other, while equation (B.5) is notably different. This was expected, because (B.7) and (B.9) are the intersections between the distortion curves at $\boldsymbol{P}$ (moving point) and the (fixed) vertex located at one side of the diagonal ( $\boldsymbol{J}$ and $\boldsymbol{K}$, respectively), whereas (B.5) is the intersection of the distortion curves at the two fixed vertices, $\boldsymbol{J}$ and $\boldsymbol{K}$. For the moment, considering just the intersection points between the distortion variation curves, we get a maximum of 9 candidate values for $m$ (if the Eqs. (B.5), (B.7) and (B.9) have three distinct real roots each), and a minimum of 3 candidates (if there is a total of three real roots only).

## Computing the local minimum

We still have to consider the $m$ candidates that come from the local minima of each curve. To do this we proceed as follows. Assume first
that $a^{2}-c\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2} \geqslant 0$, then the curve $D_{\text {Oddy }, J}(m)$ reaches local extrema at:
$m_{J}^{ \pm}=\frac{a \pm \sqrt{a^{2}-c\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}}}{\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}}$
otherwise, it reaches them at:
$m_{J}^{ \pm}=1 \pm \frac{\sqrt{-2 a+c+\left\|\boldsymbol{P}_{\boldsymbol{i}}\right\|^{2}}}{\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|}$
We now consider the case when $b^{2}-d\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2} \geqslant 0$ holds. Under this assumption the curve $D_{\text {Oddy }, K}(m)$ reaches local extrema at:
$m_{K}^{ \pm}=\frac{b \pm \sqrt{b^{2}-d\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}}}{\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}}$
otherwise, it reaches them at:
$m_{K}^{ \pm}=1 \pm \frac{\sqrt{-2 b+d+\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}}}{\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|}$
To conclude we consider the case when $(a+b)^{2}-2\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}\left(\|\boldsymbol{P} \boldsymbol{J}\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}\right) \geqslant 0$. In this case the curve $D_{\text {Oddy }, P}(m)$ reaches local extrema at:
$m_{P}^{ \pm}=\frac{a+b \pm \sqrt{(a+b)^{2}-2\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}\left(\|\boldsymbol{P} \boldsymbol{J}\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}\right)}}{2\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2}}$
otherwise, it reaches them at:
$m_{P}^{ \pm}=\frac{-2 w \pm z \sqrt{\frac{2 z\left[2 w(a+b)+z\left(\|\boldsymbol{P} \boldsymbol{J}\|^{2}+\|\boldsymbol{P} \boldsymbol{K}\|^{2}\right)\right]+4\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2} w^{2}}{\left\|\boldsymbol{P} \boldsymbol{P}_{i}\right\|^{2} z^{2}}}}{2 z}$
Collecting all the candidate values found, we are finally able to obtain the goal length for the diagonal. From Eqs. (B.5), (B.7), (B.9), (B.10), (B.11), (B.12), (B.13), (B.14) and (B.15) we obtain a set of up to 15 candidate values for $m$ ( 9 from the curves intersections if all roots are distinct and real, and 6 from the local minima). Thus we can conclude that there exists $m^{*} \in \mathbb{R}$ and $i \in\{P, J, K\}$ such that
$D_{\text {Oddy }}\left(m^{*}\right)=D_{\text {Oddy }, i}\left(m^{*}\right) \leq D_{\text {Oddy }}(m)$
for all $m \in \mathbb{R}$. This concludes the proof of the proposition.

## References

[1] S.J. Owen, A survey of unstructured mesh generation technology, in: Proceedings, 7th International Meshing Roundtable, 1998, pp. 239-267.
[2] S.H. Lo, Finite Element Mesh Generation, CRC Press, 2015.
[3] J. Sarrate, A. Huerta, Efficient unstructured quadrilateral mesh generation, Internat. J. Numer. Methods Engrg. 49 (10) (2000) 1327-1350.
[4] M. Bastian, B.Q. Li, An efficient automatic mesh generator for quadrilateral elements implemented using C++, Finite Elem. Anal. Des. 39 (9) (2003) 905-930.
[5] T.D. Blacker, M.B. Stephenson, Paving: A new approach to automated quadrilateral mesh generation, Internat. J. Numer. Methods Engrg. 32 (4) (1991) 811-847.
[6] S. Giuliani, An algorithm for continuous rezoning of the hydrodynamic grid in arbitrary Lagrangian-Eulerian computer codes, Nucl. Eng. Des. 72 (2) (1982) 205-212.
[7] A. Gargallo, X. Roca, J. Sarrate, A surface mesh smoothing and untangling method independent of the CAD parameterization, Comput. Mech. 53 (4) (2014) 587-609.
[8] P.M. Knupp, Algebraic mesh quality metrics, SIAM J. Sci. Comput. 23 (1) (2001) 193-218.
[9] G. Aparicio-Estrems, A. Gargallo-Peiró, X. Roca, Defining a stretching and alignment aware quality measure for linear and curved 2D meshes, in: Lecture Notes in Computational Science and Engineering, Springer International Publishing, 2019, pp. 37-55.
[10] A. Gargallo-Peiró, X. Roca, J. Peraire, J. Sarrate, Optimization of a regularized distortion measure to generate curved high-order unstructured tetrahedral meshes, Internat. J. Numer. Methods Engrg. 103 (5) (2015) 342-363.
[11] P. Knupp, A target construction methodology for mesh quality improvement, Eng. Comput. (2022).
[12] R. Lohner, K. Morgan, O.C. Zienkiewicz, Adaptive grid refinement for the compressible Euler equations, in: Accuracy Estimates and Adaptive Refinements in Finite Element Computations, Wiley, 1986, pp. 281-297.
[13] A. Oddy, J. Goldak, M. McDill, M. Bibby, A distortion metric for iso-parametric finite elements, CSME Trans. 12 (4) (1988) 213-217.
[14] C.K. Lee, S.H. Lo, A new scheme for the generation of a graded quadrilateral mesh, Comput. Struct. 52 (5) (1994) 847-857.
[15] S.A. Canann, J.R. Tristano, M.L. Staten, An approach to combined Laplacian and optimization-based smoothing for triangular, quadrilateral, and quad-dominant meshes, in: Proceedings of the 7th International Meshing Roundtable, 1998, pp. 479-494.
[16] J. Sarrate, A. Coll, Minimización de la distorsión de mallas formadas por cuadriláteros o hexaedros, Revista Internacional de MÉtodos NumÉricos Para CÁlculo Y DiseÑo En IngenieríA 23 (1) (2007) 55-76.
[17] S. Sholtes, Introduction to Piecewise Differentiable Equations, in: Springer Briefs in Optimization, Springer-Verlag, 2012.
[18] M. Ulbrich, Semismooth Newton Methods for Variational Inequalities and Constrained Optimization Problems in Function Spaces, in: MPS-SIAM Series on Optimization, Society for Industrial \& Applied Mathematics, 2011.
[19] J.L. Martínez, J. León, J.A. Martín-Caro, H. Corres, Análisis de la sección transversal de la Catedral de Palma de Mallorca, in: Congreso ACHE "Puentes Y Estructuras de Edificación", 2002.
[20] F.H. Clarke, Optimization and Nonsmooth Analysis, Society for Industrial and Applied Mathematics, 1990.


[^0]:    * Corresponding author at: Departamento de Matemáticas, Física y Ciencias Tecnológicas, Universidad CEU Cardenal Herrera, CEU Universities, San Bartolomé 55, 46115 Alfara del Patriarca, Spain.

    E-mail addresses: cblecua@uchceu.es (C. Blecua), afalco@uchceu.es (A. Falcó).

[^1]:    From Proposition 3.3(b) and (c) we obtain the following.

[^2]:    ${ }^{1}$ With the exception of the average distortion, which is slightly reduced.

